

A liquid-crystal model for friction

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Rate-and-state friction is an empirical approach to the behavior of a frictional surface. We use a nematic liquid crystal in a channel between two parallel planes to model frictional sliding. Nematic liquid crystals model a wide variety of physical phenomena in systems that rapidly switch between states; they are well studied and interesting examples of anisotropic non-Newtonian fluids, characterized by the orientational order of a director field $\vec{d}(\vec{x}, t)$ interacting with the velocity field $\vec{u}(\vec{x}, t)$. To model frictional sliding, we introduce a nonlinear viscosity that changes as a function of the director field orientation; the specific choice of viscosity function determines the behavior of the system. In response to sliding of the top moving plane, the fluid undergoes a rapid increase in resistance followed by relaxation. Strain is localized within the channel. The director field plays a role analogous to the state variable in rate-and-state friction.

faulting | rheology

Friictional rheologies have a wide range of applications in engineering and geophysics but are poorly understood (1). In this paper we present a nematic liquid-crystal model for friction in which the physical properties of the fluid varies with the orientation of the director field. We consider a fluid layer of prescribed thickness between two solid blocks. The blocks slide past each other at a prescribed slip velocity u_0 . The objective is to determine the resulting shear stress, which we take to be the frictional resistance. We take the fluid to be a liquid crystal that introduces a relaxation process. A fluid-based model is directly applicable to wet friction, which occurs, for example, when a layer of oil is used to lubricate two sliding surfaces; however, a fluid model can also be justified for sliding friction. Dry friction between two sliding surfaces generates granulation, resulting in the development of a granular media between the surfaces. In the case of a geologic fault, this granular material is known as fault gouge and is widely recognized (2). It is standard practice to model a shear flow in a granular material as a fluid (3).

Nematic liquid crystals are well studied examples of anisotropic non-Newtonian fluids. A liquid crystal is a phase of a material between the solid and liquid phases. The solid phase has strong intermolecular forces that keep the molecular position and orientation fixed, whereas in the liquid phase, the molecules neither occupy a specific average position nor do they remain in any particular orientation; the nematic liquid crystal phase does not have any positional order, but does possess a certain amount of orientational order. This phase is described by a velocity field, as well as a director field that describes locally the averaged direction or orientation of the constituent molecules (4).

Several empirical rate-and-state friction laws have been written to explain laboratory studies (3, 6, 7). The most widely accepted form of rate-and-state friction is the slowness law given by

$$\mu = \mu_0 + a \ln \left(\frac{u}{u_0} \right) + b \ln \left(\frac{u_0 \theta}{\mathcal{L}} \right), \quad \frac{d\theta}{dt} = 1 - \frac{\theta u}{\mathcal{L}}, \quad [1]$$

where u is slip velocity, μ is coefficient of friction, μ_0 is the reference coefficient of friction at reference velocity u_0 , θ is the

state variable, \mathcal{L} is a characteristic slip length, and a and b are parameters.

The characteristic behavior of these equations is illustrated by a step increase in the slip velocity from u_0 to u_1 . The friction coefficient increases instantaneously to the value

$$\mu_i = \mu_0 + a \ln \left(\frac{u_1}{u_0} \right). \quad [2]$$

A relaxation of the friction coefficient value then takes place to the final value given by

$$\mu = \mu_0 - (b - a) \ln \left(\frac{u_1}{u_0} \right). \quad [3]$$

If $b > a$ the final coefficient of friction decreases with increasing velocity. This is velocity weakening and leads to the stick-slip behavior associated with faults.

Liquid-Crystal Model

We use a liquid-crystal fluid, flowing in a horizontal channel between two parallel plates, to model frictional sliding (Fig. 1). Liquid crystals are extensively studied and have applications to a wide variety of engineered systems, including systems that rapidly switch between states (4). The liquid crystal is characterized by a directional field, $\vec{d}(y, t)$, where y is the vertical distance from the fixed lower plate and t is time.

The viscosity is given by $\nu = \alpha(\theta)v_1 + (1 - \alpha(\theta))v_0$, where θ is the angle of the director field \vec{d} with respect to the vertical. The minimum viscosity v_0 occurs when \vec{d} points in the horizontal direction, whereas the maximum viscosity v_1 occurs when \vec{d} is vertical. The choice of the function $\alpha(\theta)$ determines the type of friction that we simulate.

A third parameter γ denotes the relaxation coefficient of the director \vec{d} . When the velocity of the upper plate \vec{u} is suddenly increased, the fluid undergoes a rapid increase in resistance, followed by a relaxation. The director \vec{d} is deflected from the vertical, and strain is localized near the center of the channel. The director field \vec{d} plays a role analogous to the state variable in rate-and-state friction. Reducing the relaxation coefficient γ of \vec{d} produces a sharper increase in traction change as a function of velocity, but when γ is very small, numerical instability can enter the simulation. Reducing the minimum viscosity v_0 to 0 produces stick-slip-like behavior, but restricts the choice of the function $\alpha(\theta)$; the choice of $\alpha(\theta)$, in turn, controls both the size of the traction jump associated with changes in velocity and the resulting relaxation back to the equilibrium state.

The model is described by the following equations of motion. Conservation of momentum requires that

$$\vec{u}_t + (\vec{u} \cdot \nabla)\vec{u} = \text{div}(\nu \nabla \vec{u}) - \frac{1}{\rho} \nabla p \quad \text{in } (0, T) \times \Omega, \quad [4]$$

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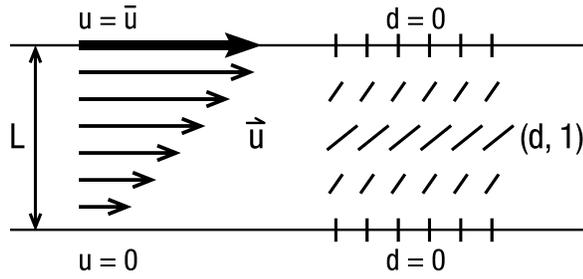


Fig. 1. Illustration of our fault model, a horizontal channel of thickness L filled with a liquid-crystal. The horizontal velocity u increases from $u = 0$ at the base to $u = \bar{u}$ at the top. The director field has a constant vertical component; the horizontal component is zero at the boundaries and increases to a maximum at the center.

Where \bar{u} is the velocity, ρ is density, ν is kinematic viscosity, and p is pressure, Ω denotes either a two- or three-dimensional smooth open set, and $[0, T]$ denotes the interval of time t on which we study the evolution of this system.

We assume that the fluid is incompressible, so that

$$\operatorname{div} \vec{u} = 0 \quad \text{in } [0, T] \times \Omega. \quad [5]$$

The viscosity in the fluid is determined by the orientation of the vector director field, \vec{d} . The director field behaves in a manner analogous to the velocity:

$$\vec{d}_t + \nabla \vec{d} \cdot \vec{u} - \nabla \vec{u} \cdot \vec{d} = \gamma \Delta \vec{d} \quad \text{in } (0, T) \times \Omega, \quad [6]$$

where γ is a relaxation parameter that plays a role analogous to the viscosity in Eq. 4. Note that there is an interaction between the velocity field \vec{u} , which changes the orientation of the director field \vec{d} ; the director field feeds back to the velocity field through its influence on viscosity.

The viscosity of the fluid ν depends on a function α , which depends only on the angle θ between the director field and the fluid velocity. We choose a relationship between ν and α that is based on the behavior of liquid crystals (8) but modified to yield a friction-like behavior.

$$\nu = \alpha(\theta)v_1 + (1 - \alpha(\theta))v_0, \quad \text{in } (0, T) \times \Omega, \quad [7]$$

where $\cos \theta = \|\vec{u} \times \vec{d}\| / (\|\vec{u}\| \|\vec{d}\|)$. The required initial and boundary conditions are taken to be

$$\vec{u} = \vec{u}_{BC} \quad \text{on } [0, T] \times \partial\Omega, \quad [8]$$

$$\vec{d} = \vec{d}_{BC} \quad \text{on } [0, T] \times \partial\Omega, \quad [9]$$

$$\vec{u} = \vec{u}_{ic} \quad \text{on } \{t = 0\} \times \Omega, \quad [10]$$

$$\vec{d} = \vec{d}_{ic} \quad \text{on } \{t = 0\} \times \Omega, \quad [11]$$

where subscript BC and ic denote specified boundary and initial conditions.

To use this model to characterize the behavior of a fault, we consider a two-dimensional infinite channel so that $\Omega = (-\infty, \infty) \times (0, L)$ with coordinates (x, y) . The channel is filled with a fluid and we take the horizontal pressure gradient to be zero so that the flow is driven by the imposed velocity of the upper boundary of the channel \vec{u} .

We let $\vec{u}(x, y) = (u_1(x, y), u_2(x, y))$ and similarly $\vec{d}(x, y) = (d_1(x, y), d_2(x, y))$. Proceeding with the usual channel flow assumptions, we assume that the vertical component of the velocity field vanishes so that $u_2 = 0$, and that the horizontal component only depends on the vertical coordinate y . We set $u(y) := u_1(y)$. For

the channel geometry, we set $d_2 = 1$ and assume that the horizontal component d_1 only depends on the vertical coordinate so that we now take

$$d(y) := d_1(y) \quad [12]$$

The model is illustrated in Fig. 1.

With these assumptions, and using the notation $u' = \frac{\partial u}{\partial y}$, the full system of equations simplifies as follows:

$$u_t = (\nu u')' \quad \text{in } (0, T) \times (0, L) \quad [13]$$

$$d_t = \gamma d'' + u' \quad \text{in } (0, T) \times (0, L) \quad [14]$$

$$\cos \theta = (1 + d^2)^{-1/2} \quad \text{in } (0, T) \times (0, L) \quad [15]$$

$$\nu = \alpha(\theta)v_1 + (1 - \alpha(\theta))v_0, \quad \text{in } (0, T) \times (0, L) \quad [16]$$

$$u(0, t) = 0, u(L, t) = \bar{u} \quad t \geq 0 \quad [17]$$

$$d(0, t) = d(L, t) = 0 \quad t \geq 0 \quad [18]$$

$$u(y, 0) = \frac{\bar{u}(0)}{L}y \quad 0 \leq y \leq L \quad [19]$$

$$d(y, 0) = 0 \quad 0 \leq y \leq L \quad [20]$$

where $\bar{u}(0)$ is the initial sliding velocity of the top plate.

To simplify our analysis, we introduce nondimensional variables. Our reference length is the channel width L , our reference velocity is the initial prescribed velocity of the upper boundary $\bar{u}(0)$; the reference viscosity is the viscosity at the boundaries v_1 where $\theta = 0$, and the reference time is $L/\bar{u}(0)$. The derived nondimensional parameters are the Reynolds number:

$$\mathbf{Re} = \frac{\bar{u}(0)L}{v_1},$$

which governs the viscous behavior, and the director number,

$$D = \frac{v_1}{\gamma}.$$

The director number D is the ratio of the diffusion coefficient for vorticity, the kinematic viscosity v_1 , to the diffusion coefficient for the director field, γ . It is also the ratio of the relaxation time for the two processes. If $D \gg 1$ the velocity field relaxes in a much shorter time than the director field. If $D \ll 1$, the director field relaxes in a much shorter time than the velocity field. Because D is the ratio of two relaxation times, it resembles, but is not equivalent to, either the Deborah number (9), which is the ratio of the relaxation time of a viscous material to the timescale for observation, or the Weissenberg number, the ratio of a relaxation time to a process time for a viscoelastic material.

Let $u \mapsto \bar{u}(0)u$; $\bar{u} \mapsto \bar{u}(0)\bar{u}$; $y \mapsto Ly$; $t \mapsto Lt/\bar{u}(0)$, $d \mapsto d_0d$ and $\nu \mapsto \nu_1\nu$ be the changes of variables; then Eqs. 13 to 20 become

$$u_t = \frac{1}{\mathbf{Re}}(\nu u')' \quad \text{in } (0, T) \times (0, 1) \quad [21]$$

$$d_t = \frac{1}{D\mathbf{Re}}d'' + u' \quad \text{in } (0, T) \times (0, 1) \quad [22]$$

$$\cos \theta = (1 + d^2)^{-1/2} \quad \text{in } (0, T) \times (0, 1) \quad [23]$$

$$\nu = \alpha(\theta)v_1 + (1 - \alpha(\theta))v_0, \quad \text{in } (0, T) \times (0, 1) \quad [24]$$

$$u(0, t) = 0, u(1, t) = \bar{u} \quad t \geq 0 \quad [25]$$

$$d(0, t) = d(1, t) = 0 \quad t \geq 0 \quad [26]$$

$$u(y, 0) = y \quad 0 \leq y \leq 1 \quad [27]$$

$$d(y, 0) = 0 \quad 0 \leq y \leq 1 \quad [28]$$

We define a smooth transition function α from the minimum to maximum states of viscosity in Eq. 16. We choose

$$\alpha(\theta) = \begin{cases} 1 & \text{if } 0.9 \leq \cos \theta \leq 1 \\ \frac{e^{10 \cos \theta} - 1}{e^9 - 1} & \text{if } 0 \leq \cos \theta \leq 0.9, \end{cases} \quad [29]$$

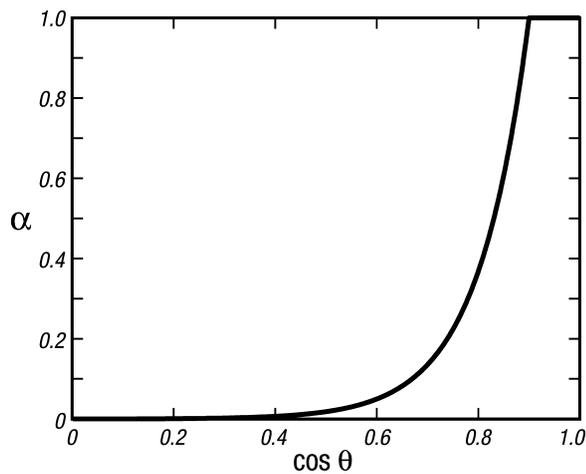


Fig. 2. Dependence of the transition function α for the viscosity as defined in Eq. 16 on the angle of the director field θ .

and this dependence is illustrated in Fig. 2. The model is now fully prescribed.

Simulations

We first give the results in the steady state. The shear stress (traction) τ is a constant across the channel. Using our nondimensionalization, we have $\tau \mapsto v_1 \bar{u}(0) \tau / L$. We give the dependence of the nondimensional traction τ and the maximum value of the horizontal component of the director field d_{\max} on the Reynolds number in Fig. 3 for two values of the director number $D = 30$ and $D = 300$. The dependence on the Reynolds number is equivalent to a dependence on the velocity of the upper plate \bar{u} . In both

cases we have velocity strengthening for small Re (small \bar{u}) and velocity weakening for large Re (large \bar{u}). The velocity weakening behavior is similar to that of rate and state friction but without the singular behavior at $\bar{u} = 0$.

The maximum of the director d indicates the largest angle the director is deflected by the fluid. In both cases, it increases linearly initially, while starting to increase more rapidly when it is bigger than 0.5. Note that when $d > 0.5$, $\cos \theta = (1 + d^2)^{-1/2}$ is smaller than 0.9, in which case α starts to drop dramatically. When this occurs, τ , the traction at the top, decreases accordingly.

The laboratory experiments used to derive rate-and-state friction laws use a sudden increase of the sliding velocity from $\bar{u}(0)$ to $\bar{u}(0) + \delta u$ and then a sudden decrease back to $\bar{u}(0)$ (5–7, 10). We next study the transient response of the liquid-crystal layer to a rapid change in the top-plate sliding velocity \bar{u} . In the following simulations, a tanh(tan)-type of transition is considered: the top plate sliding velocity is defined as

$$\bar{u} = \begin{cases} 1 & \text{if } t \in [0, 14.9], \\ \mathbf{u} + \frac{\delta u}{2} \tanh[\tan(5(t - 15)\pi)] & \text{if } t \in [14.9, 15.1], \\ 1 + \delta u & \text{if } t \in [15.1, 19.9], \\ \mathbf{u} - \frac{\delta u}{2} \tanh[\tan(5(t - 20)\pi)] & \text{if } t \in [19.9, 20.1], \\ 1 & \text{if } t \in [20.1, 25], \end{cases} \quad [30]$$

where $\mathbf{u} = 1 + \delta u/2$. The change of the top-plate sliding velocity is shown in Fig. 4. This dependence is a close approximation to a step increase followed by a step decrease in the upper plate velocity.

We give numerical solutions for two examples:

Case 1: $\text{Re} = 0.04129$, $D = 300$ and $\delta u = 0.0145$.

Case 2: $\text{Re} = 0.02443$, $D = 30$ and $\delta u = 0.0914$.

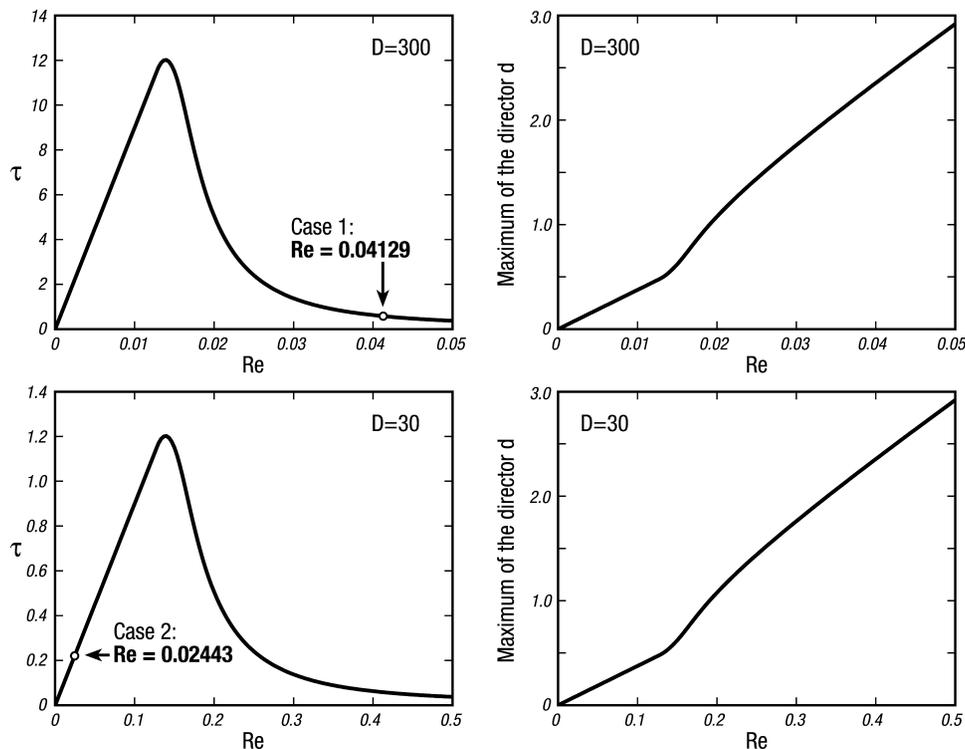


Fig. 3. Steady-state dependence of the nondimensional traction τ and the maximum horizontal value of the traction field d on the Reynolds number Re . Two values of the director number are considered: $D = 300$ and $D = 30$. Also shown are the Reynolds numbers of the two transient solutions that we present.

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